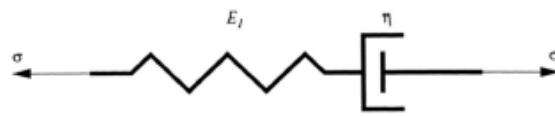


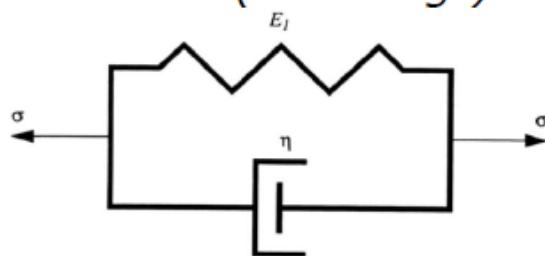
# Simple Viscoelastic Models

Stress depends on strain *and* strain-rate:

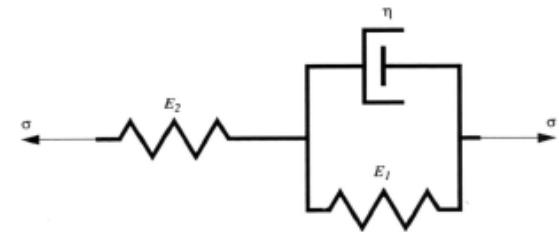
Maxwell



Kelvin (aka Voigt)



Standard Linear Solid



- Elastic stress depends on strain (spring)
- Viscous stress depends on strain-rate (damper)
- Maxwell: Strains add in series, stresses are equal
- Kelvin: Stresses add in parallel, strains are equal
- SLS: Combination of Maxwell and Kelvin

## Maxwell Model



Total strain = spring strain + dashpot strain:

$$\varepsilon = \varepsilon_{E_1} + \varepsilon_\eta \quad \Rightarrow \dot{\varepsilon} = \dot{\varepsilon}_{E_1} + \dot{\varepsilon}_\eta$$

$$\sigma = \sigma_{E_1} = \sigma_\eta \quad \sigma_{E_1} = E_1 \cdot \varepsilon_{E_1} \quad \sigma_\eta = \eta \cdot \dot{\varepsilon}_\eta$$

$$\Rightarrow \dot{\varepsilon}_{E_1} = \frac{\dot{\sigma}}{E_1}$$

$$\dot{\varepsilon}_\eta = \frac{\sigma}{\eta}$$

$$\boxed{\dot{\varepsilon} = \frac{\dot{\sigma}}{E_1} + \frac{\sigma}{\eta}}$$

A linear first-order ordinary differential equation (ODE)

## Creep Solution

Does the model creep?

$$\dot{\varepsilon} = \frac{\dot{\sigma}}{E_1} + \frac{\sigma}{\eta}$$

$$\frac{d\varepsilon}{dt} = \frac{1}{E_1} \frac{d\sigma}{dt} + \frac{\sigma}{\eta}$$

$$d\varepsilon = \frac{1}{E_1} d\sigma + \frac{\sigma}{\eta} dt$$

Integrating, for constant applied stress,  $\sigma_0$  :

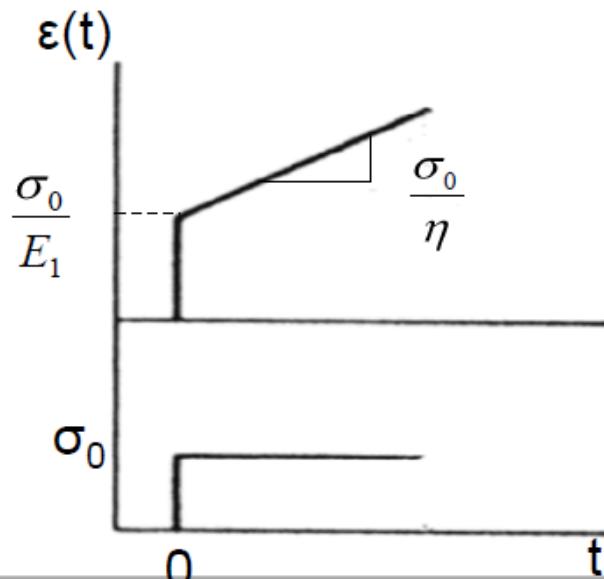
$$\int_{\varepsilon(0)}^{\varepsilon(t)} d\varepsilon = \frac{1}{E_1} \cancel{\int_{\sigma(0)}^{\sigma(t)} d\sigma} + \frac{\sigma_0}{\eta} \int_0^t dt + C \Rightarrow \varepsilon(t) = \varepsilon(0) + \frac{\sigma_0 t}{\eta} + C$$

## Creep Solution

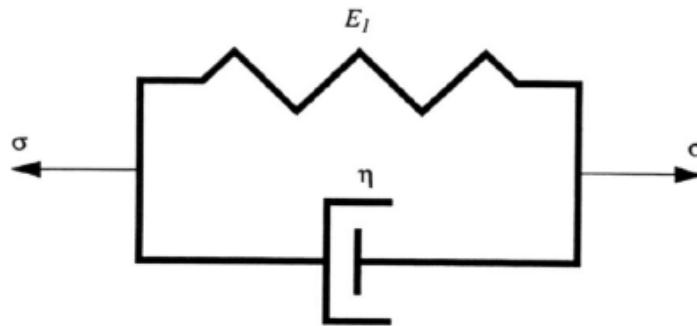
$$\varepsilon(t) = \varepsilon(0) + \frac{\sigma_0 t}{\eta} + C$$

Only the Hooke element reacts initially:  $\varepsilon(0) = C = \frac{\sigma_0}{E_1}$

$$\Rightarrow \varepsilon(t) = \frac{\sigma_0}{E_1} + \frac{\sigma_0 t}{\eta} = \sigma_0 \left( \frac{1}{E_1} + \frac{t}{\eta} \right) = \sigma_0 \underbrace{J(t)}_{\text{Creep function}}$$



## Kelvin Model



Total stress = spring stress + dashpot stress:

$$\sigma = \sigma_{E_1} + \sigma_\eta \quad \varepsilon = \varepsilon_{E_1} = \varepsilon_\eta$$

$$\sigma_{E_1} = E_1 \cdot \varepsilon \quad \sigma_\eta = \eta \cdot \dot{\varepsilon}$$

$$\boxed{\sigma = E_1 \cdot \varepsilon + \eta \cdot \dot{\varepsilon}}$$

A linear first-order ordinary differential equation (ODE)

## Creep Solution

$$\sigma = E_1 \cdot \varepsilon + \eta \cdot \dot{\varepsilon}$$

Does the model creep?

Constant stress,  $\sigma_0$ :  $\frac{d\varepsilon}{dt} + \frac{E_1}{\eta} \varepsilon = \frac{\sigma_0}{\eta}$        $\varepsilon(t) = \varepsilon_H(t) + \varepsilon_N(t)$

$$\frac{d\varepsilon_H}{dt} + \frac{E_1}{\eta} \varepsilon_H = 0 \Rightarrow \frac{d\varepsilon_H}{\varepsilon_H} = -\frac{E_1}{\eta} dt \Rightarrow \varepsilon_H(t) = C_1 e^{-\frac{E_1}{\eta} t}$$

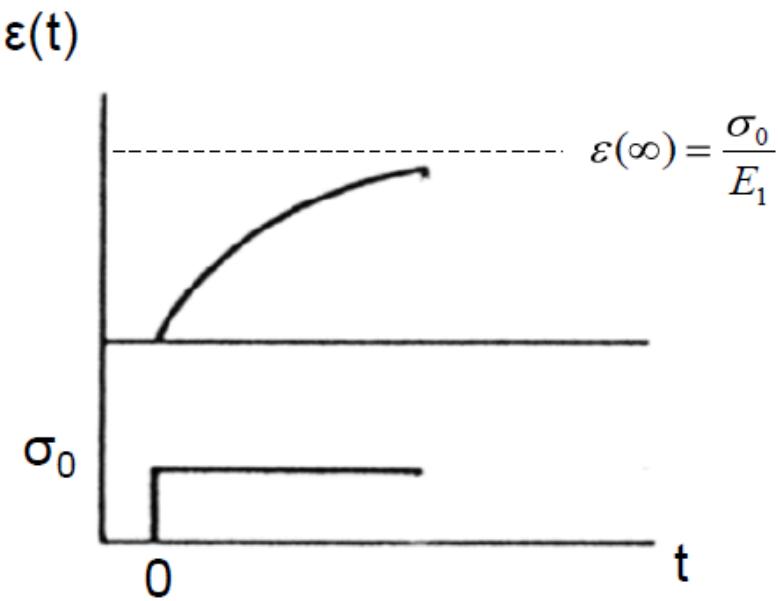
$$\varepsilon_N(t) = C_2 \Rightarrow \varepsilon(t) = C_1 e^{-\frac{E_1}{\eta} t} + C_2 \Rightarrow \frac{d\varepsilon(t)}{dt} = -\frac{E_1}{\eta} C_1 e^{-\frac{E_1}{\eta} t}$$

$$-\frac{E_1}{\eta} C_1 e^{-\frac{E_1}{\eta} t} + \frac{E_1}{\eta} C_1 e^{-\frac{E_1}{\eta} t} + \frac{E_1}{\eta} C_2 = \frac{\sigma_0}{\eta} \Rightarrow C_2 = \frac{\sigma_0}{E_1}$$

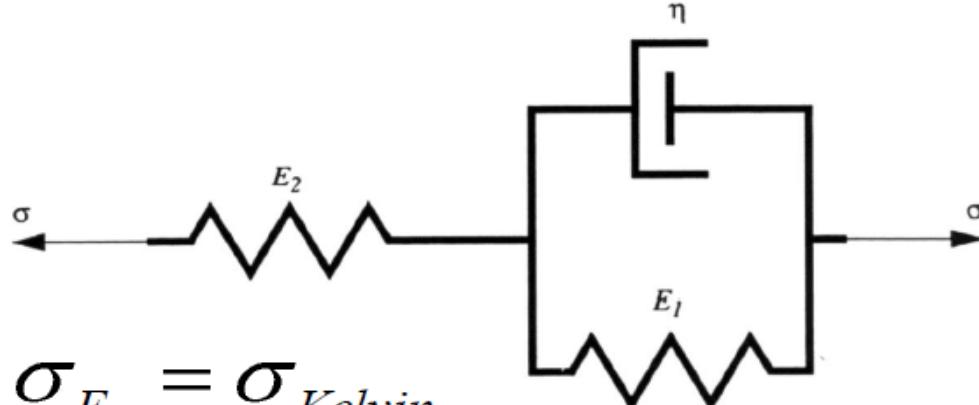
$$\Rightarrow \varepsilon(t) = C_1 e^{-\frac{E_1}{\eta} t} + \frac{\sigma_0}{E_1}$$

# Creep Solution

$$\varepsilon(0) = 0 \Rightarrow C_1 = -\frac{\sigma_0}{E_1} \quad \Rightarrow \varepsilon(t) = \frac{\sigma_0}{E_1} \left(1 - e^{-\frac{E_1}{\eta}t}\right) = \sigma_0 J(t)$$



## Standard Linear Solid



$$\sigma = \sigma_{E_2} = \sigma_{\text{Kelvin}}$$

$$\varepsilon = \varepsilon_{E_2} + \varepsilon_{\text{Kelvin}}$$

$$\varepsilon_{E_2} = \frac{\sigma}{E_2} \Rightarrow \dot{\varepsilon}_{E_2} = \frac{\dot{\sigma}}{E_2}$$

$$\boxed{\sigma_{\text{Kelvin}} = E_1 \cdot \varepsilon_{\text{Kelvin}} + \eta \cdot \dot{\varepsilon}_{\text{Kelvin}}}$$

$$\dot{\varepsilon} = \dot{\varepsilon}_{E_2} + \dot{\varepsilon}_{\text{Kelvin}}$$

$$\Rightarrow \dot{\varepsilon}_{\text{Kelvin}} = (\sigma - E_1 \cdot \varepsilon_{\text{Kelvin}}) / \eta$$

## Standard Linear Solid

$$\Rightarrow \dot{\varepsilon} = \frac{\dot{\sigma}}{E_2} + (\sigma - E_1 \cdot \varepsilon_{kelvin}) / \eta$$

$$\varepsilon_{kelvin} = \varepsilon - \varepsilon_{E_2} \Rightarrow$$

$$\dot{\varepsilon} = \frac{\dot{\sigma}}{E_2} + \frac{\sigma}{\eta} - \frac{E_1}{\eta} \cdot (\varepsilon - \varepsilon_{E_2}) \Rightarrow$$

$$\varepsilon_{E_2} = \frac{\sigma}{E_2} \Rightarrow$$

$$\boxed{\dot{\varepsilon} = \frac{\dot{\sigma}}{E_2} + \frac{\sigma}{\eta} \left(1 + \frac{E_1}{E_2}\right) - \frac{E_1}{\eta} \varepsilon}$$

## Creep Solution

Does the model creep?

$$\dot{\varepsilon} = \frac{\dot{\sigma}}{E_2} + \frac{\sigma}{\eta} \left(1 + \frac{E_1}{E_2}\right) - \frac{E_1}{\eta} \varepsilon$$

A constant stress,  $\sigma_0$  is instantaneously applied at time  $t=0$ ,  
Constant stress  $\rightarrow d\sigma/dt=0$ , when  $t>0$

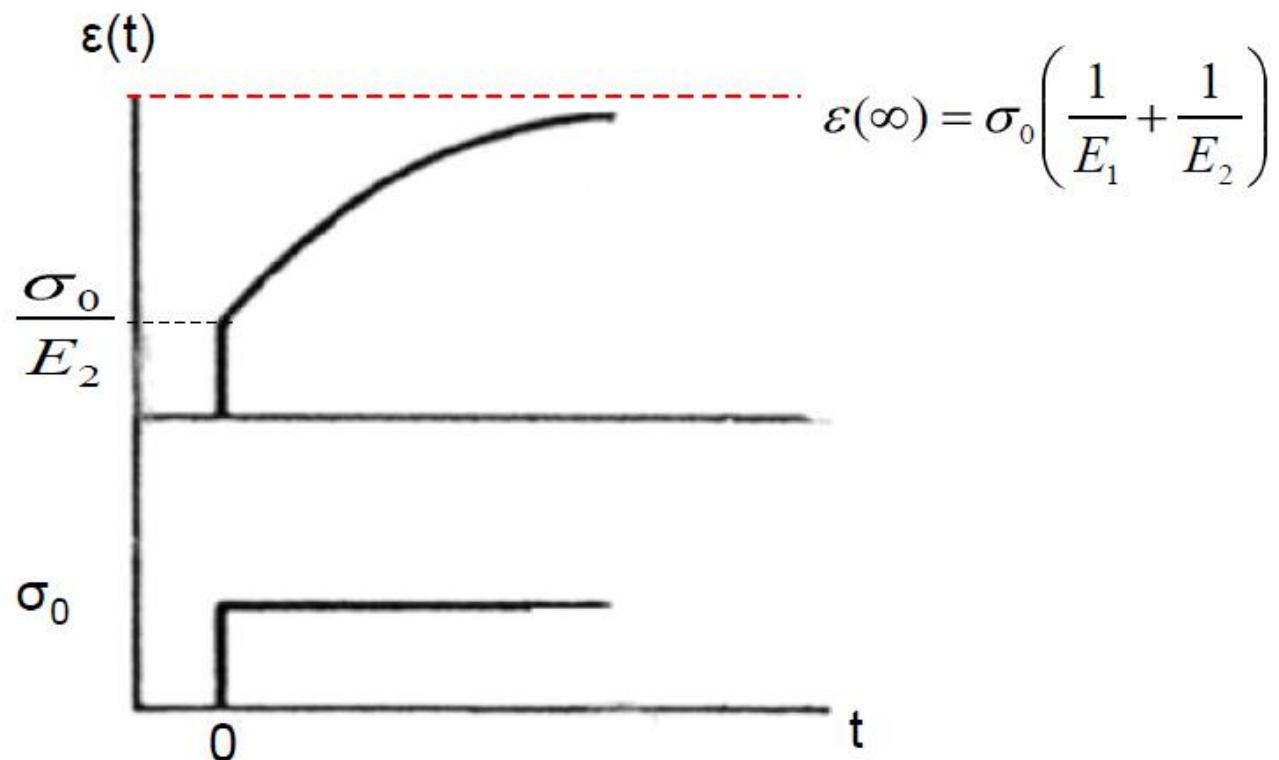
$$\frac{d\varepsilon}{dt} + \frac{E_1}{\eta} \varepsilon = \frac{\sigma_0}{\eta} \left(1 + \frac{E_1}{E_2}\right) \Rightarrow$$

A linear first-order ordinary differential equation (ODE)

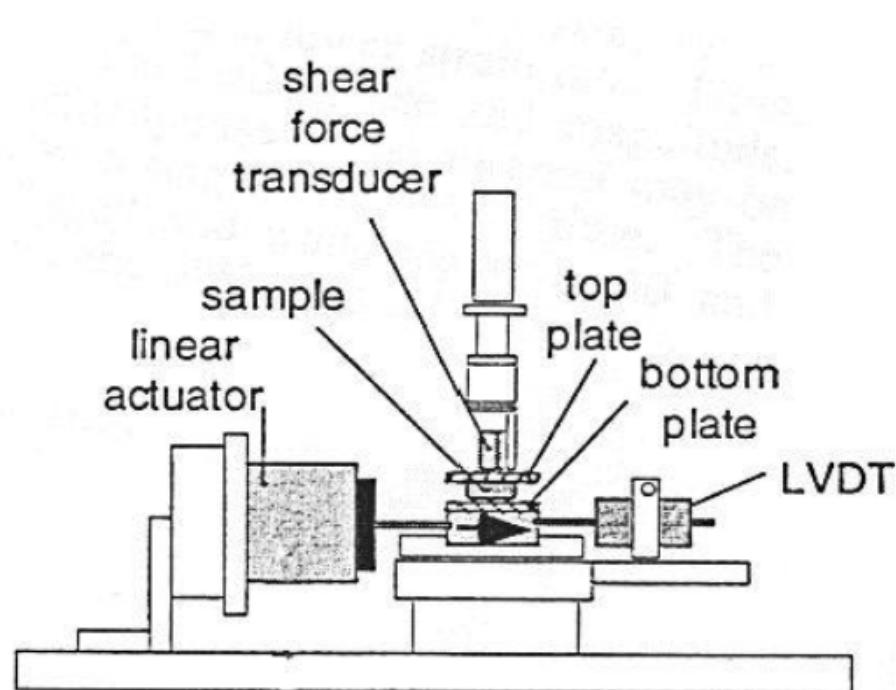
Solving, and using the fact that only  
the Hooke element reacts initially  $\Rightarrow \varepsilon(0) = \frac{\sigma_0}{E_2} \Rightarrow$

## Creep Solution

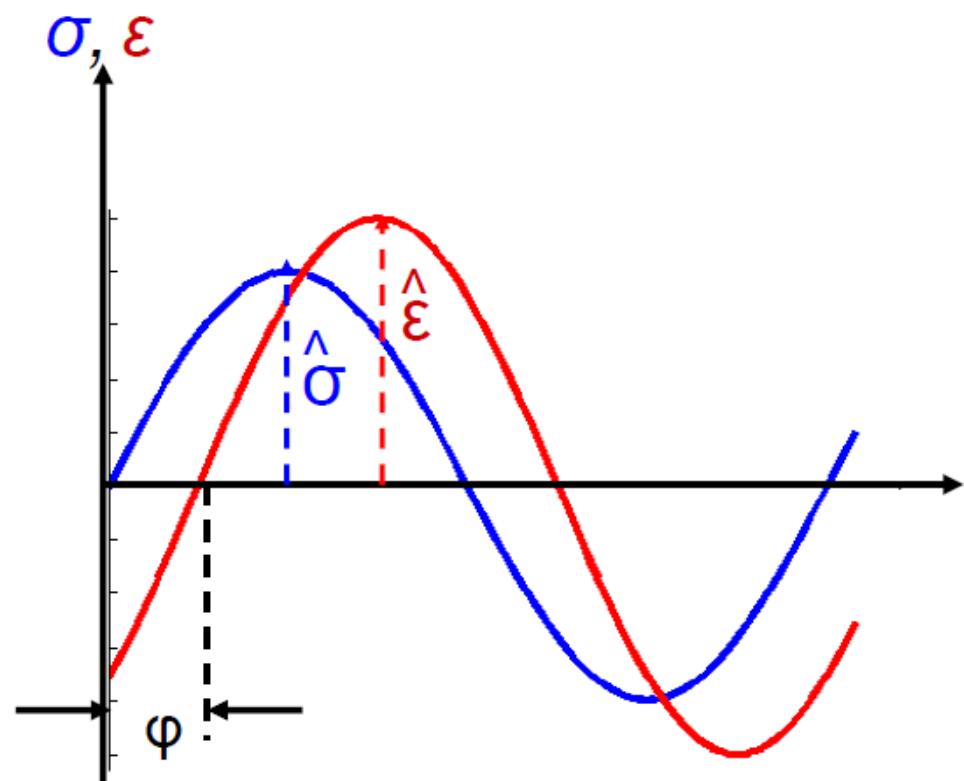
$$\dots \Rightarrow \varepsilon(t) = \sigma_0 \left[ \frac{1}{E_1} \left( 1 - e^{-\frac{E_1 t}{\eta}} \right) + \frac{1}{E_2} \right] = \sigma_0 J(t)$$



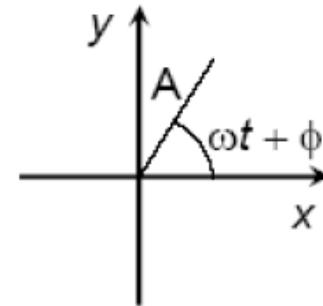
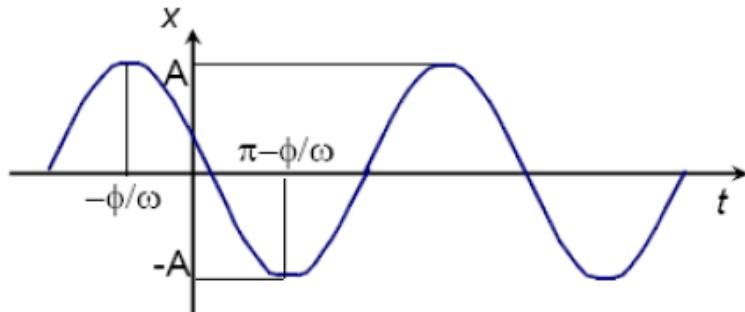
# Oscillations to determine viscoelastic properties



$$\varepsilon = \hat{\varepsilon} \sin(\omega t)$$



# Harmonic Loading



$$y = A \sin(\omega t + \phi)$$
$$x = A \cos(\omega t + \phi)$$

$$\begin{aligned} z = x + iy &= A \cos(\omega t + \phi) + i \sin(\omega t + \phi) \\ &= Ae^{i(\omega t + \phi)} = Ae^{i\phi} e^{i\omega t} = Be^{i\omega t} \end{aligned}$$

$A = \text{amplitude} = |B|$

$\phi = \text{phase angle} = \tan^{-1}(\text{Im } B / \text{Re } B)$

$\omega = \text{angular frequency} \quad 2\pi f \quad (\text{rad/s})$

## Harmonic Stress and Strain History

$$\varepsilon = \hat{\varepsilon} \cos(\omega t)$$

$$\varepsilon = \hat{\varepsilon} \cdot e^{i\omega t}$$

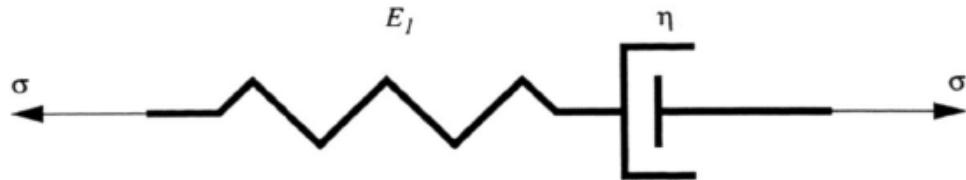
$$\dot{\varepsilon} = i\omega \hat{\varepsilon} \cdot e^{i\omega t}$$

$$\Rightarrow \dot{\varepsilon} = i\omega \varepsilon$$

$$\sigma = \hat{\sigma} \cos(\omega t)$$

$$\dots \Rightarrow \dot{\sigma} = i\omega \sigma$$

Maxwell Model



$$\dot{\varepsilon} = \frac{\dot{\sigma}}{E_1} + \frac{\sigma}{\eta}$$

$$\Rightarrow i\omega \varepsilon = \sigma \left( \frac{i\omega}{E_1} + \frac{1}{\eta} \right)$$

$$\Rightarrow \sigma(i\omega) = \underbrace{\left( \frac{i\omega}{E_1} + \frac{1}{\eta} \right)^{-1}}_{E(i\omega)} \varepsilon$$

$$E(i\omega)$$

## Complex modulus for the Maxwell Model

$$\Rightarrow E(i\omega) = i\omega \frac{1}{\frac{i\omega}{E_1} + \frac{1}{\eta}} \times \left( \frac{i\omega}{E_1} - \frac{1}{\eta} \right)$$
$$\Rightarrow E(i\omega) = \frac{\frac{E_1}{\eta}}{-\frac{\omega^2}{E_1^2} - \frac{i\omega}{\eta^2}}$$

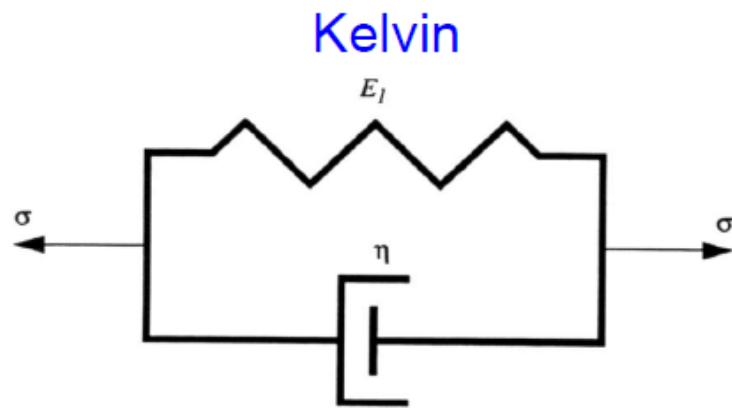
$$\Rightarrow E(i\omega) = \left( \frac{\omega^2}{E_1} + \frac{i\omega}{\eta} \right) / \left( \frac{\omega^2}{E_1^2} + \frac{1}{\eta^2} \right)$$

**Re  $E(i\omega)$**       **Im  $E(i\omega)$**

$$E(i\omega) = E' + iE''$$

"Stiffness"      "Damping"

## Complex modulus for the Kelvin Model



$$\sigma = E_1 \cdot \varepsilon + \eta \cdot \dot{\varepsilon}$$

$$\dot{\varepsilon} = i\omega\varepsilon \Rightarrow$$

$$\sigma(i\omega) = (\underbrace{E_1 + i\omega \cdot \eta}_{E(i\omega)})\varepsilon$$

$$E(i\omega) = \boxed{E_1} + \boxed{i\omega \cdot \eta}$$

$$\boxed{\operatorname{Re} E(i\omega)}$$

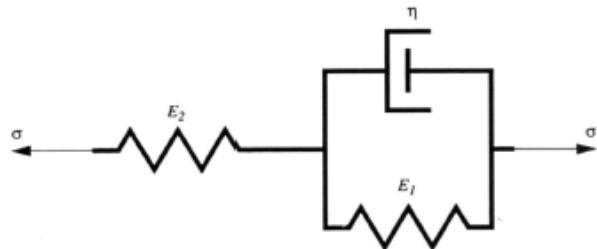
$$\boxed{\operatorname{Im} E(i\omega)}$$

"Stiffness"

"Damping"

## Complex modulus for the SLS Model

Standard Linear Solid



$$\dot{\varepsilon} = i\omega\varepsilon$$

$$\dot{\sigma} = i\omega\sigma$$

$$\dot{\varepsilon} = \frac{\dot{\sigma}}{E_2} + \frac{\sigma}{\eta} \left(1 + \frac{E_1}{E_2}\right) - \frac{E_1}{\eta} \varepsilon$$

$$i\omega\eta\sigma + (E_1 + E_2)\sigma = i\omega E_2\eta\varepsilon + E_1 E_2 \varepsilon \Rightarrow$$

$$\sigma = \frac{E_1 + i\omega\eta}{\underbrace{(E_1 + E_2) + i\omega\eta}_{E(i\omega)}} E_2 \cdot \varepsilon$$

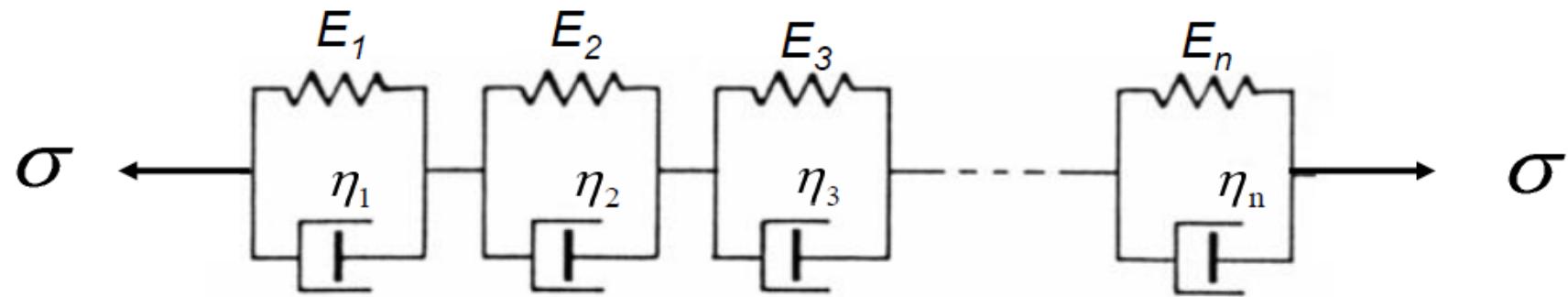
$$E(i\omega) = \frac{E_1 + i\omega\eta}{(E_1 + E_2) + i\omega\eta} E_2 \Rightarrow \dots$$

$$E(i\omega) = \frac{E_1 E_2 (E_1 + E_2) + (\omega\eta)^2 E_2}{(E_1 + E_2)^2 + (\omega\eta)^2} + i \frac{\omega\eta E_2^2}{(E_1 + E_2)^2 + (\omega\eta)^2}$$

What if the curve of the model does not fit the curve of the material we want to describe?

Generalized Models...

## Generalized Kelvin Model



Total strain =  $\sum$  (strains in each Kelvin element)

$$\varepsilon = \varepsilon_1 + \varepsilon_2 + \dots + \varepsilon_n = \sum_{i=1}^n \varepsilon_i$$

Creep function:

$$\varepsilon(t) = \sum_{i=1}^n \frac{\sigma_0}{E_i} \left(1 - e^{-\frac{E_i t}{\eta_i}}\right)$$

# The Convolution Integral for Stress

